

FORM PTO-1390

U.S. DEPARTMENT OF COMMERCE PATENT AND TRADEMARK OFFICE
TRANSMITTAL LETTER TO THE UNITED STATES
DESIGNATED/ELECTED OFFICE (DO/EO/US)
CONCERNING A FILING UNDER 35 U.S.C. 371

ATTORNEY'S DOCKET NUMBER:
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U.S. APPLICATION NUMBER (See 37 CFR 1.5)
10/019585

INTERNATIONAL APPLICATION NO.:
PCT/SE00/01394

INTERNATIONAL FILING DATE:
30 JUNE 2000 (30.06.00)

PRIORITY DATE CLAIMED:
30 JUNE 1999 (30.06.99)

TITLE OF INVENTION: OIL SEPARATOR FOR SMALL PARTS

APPLICANT(S) FOR DO/EO/US: Nils Olof HÅKANSSON and Per PERSSON

Applicant herewith submits to the United States Designated/Elected Office (DO/EO/US) the following items and other information:

1. ☒ This is a **FIRST** submission of items concerning a filing under 35 U.S.C. 371.
 2. ☐ This is a **SECOND** or **SUBSEQUENT** submission of items concerning a filing under 35 U.S.C. 371.
 3. ☒ This express request to begin national examination procedures (35 U.S.C. 371(f)) at any time rather than delay examination until the expiration of the applicable time limit set in 35 U.S.C. 371(b) and PCT Articles 22 and 39(1).
 4. ☒ A proper Demand for International Preliminary Examination was made by the 19th month from the earliest claimed priority date.
 5. ☒ A copy of the International Application as filed (35 U.S.C. 371(c)(2))
 - a. ☒ is transmitted herewith (required only if not transmitted by the International Bureau).
 - b. ☐ has been transmitted by the International Bureau. (see attached copy of PCT/IB/308)
 - c. ☐ is not required, as the application was filed in the United States Receiving Office (RO/US).
 6. ☐ A translation of the International Application into English (35 U.S.C. 371(c)(2)).
 7. ☐ Amendments to the claims of the International Application under PCT Article 19 (35 U.S.C. 371(c)(3)).
 - a. ☐ are transmitted herewith (required only if not transmitted by the International Bureau).
 - b. ☐ have been transmitted by the International Bureau.
 - c. ☐ have not been made; however, the time limit for making such amendments has NOT expired.
 - d. ☐ have not been made and will not be made.
 8. ☐ A translation of the amendments to the claims under PCT Article 19 (35 U.S.C. 371(c)(3)).
 9. ☐ An oath or declaration of the inventor(s) (35 U.S.C. 371(c)(4)).
 10. ☐ A translation of the annexes of the International Preliminary Examination Report under PCT Article 36 (35 U.S.C. 371(c)(5)).
- Item 11. to 16. below concern document(s) or information included:
11. ☒ An Information Disclosure Statement under 37 CFR 1.97 and 1.98.
 12. ☐ An assignment document for recording. A separate cover sheet in compliance with 37 CFR 3.28 and 3.31 is included.
 13. ☒ A **FIRST** preliminary amendment.
 14. ☐ A **SECOND** or **SUBSEQUENT** preliminary amendment.
 15. ☐ A substitute specification.
 16. ☐ A change of power of attorney and/or address letter.
 16. ☒ Other items or information: INTERNATIONAL PRELIMINARY EXAMINATION REPORT (PCT/IPEA/409), INTERNATIONAL PRELIMINARY EXAMINATION REPORT (PCT/ISA/210), APPLICATION DATA SHEET, ABSTRACT

17.	X	The following fees are submitted:
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BASIC NATIONAL FEE (37 CFR 1.492(a)(1)-(5)):

Neither international preliminary examination fee (37 CFR1.482) nor international search fee (37 CFR1.445(a)(2)) paid to USPTO and International Search Report not prepared by the EPO or JPO

International preliminary examination fee (37 CFR 1.482) not paid to USPTO but International Search Report prepared by the EPO or JPO	\$ 890.00
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International preliminary examination fee (37 CFR 1.482) not paid to USPTO but international search fee (37 CFR 1.445(a)(2)) paid to USPTO	\$ 740.00
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International preliminary examination fee (37 CFR 1.482) paid to USPTO but all claims did not satisfy provisions of PCT Article 33(1)-(4)	\$ 710.00
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International preliminary examination fee (37 CFR 1.482) paid to USPTO and all claims satisfied provisions of PCT Article 33(1)-(4)	\$ 100.00
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ENTER APPROPRIATE BASIC FEE AMOUNT =

Surcharge of \$130.00 for furnishing the oath or declaration later than 30 months from the earliest claimed priority date (37 CFR 1.492(e)).

CLAIMS	NUMBER FILED	NUMBER EXTRA	RATE
Total claims	8 - 20 =	0	X \$18.00
Independent claims	1 - 3 =	0	X \$84.00
MULTIPLE DEPENDENT CLAIMS(S) (if applicable)			+ \$280.00

TOTAL OF ABOVE CALCULATIONS =

Reduction of 1/2 for filing by small entity, if applicable. Applicant claims Small Entity Status under 37 CFR 1.27. +

SUBTOTAL =

Processing fee of \$130 for furnishing the English translation later than months from the earliest claimed priority date (37 CFR1.492(f)).

TOTAL NATIONAL FEE =

Fee for recording the enclosed assignment (37 CFR1.21(h)). The assignment must be accompanied by an appropriate cover sheet (37 CFR 3.28, 3.31). \$40.00 per property

TOTAL FEES ENCLOSED =

Amount to be refunded:

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- | | | |
|----|-------------------------------------|--|
| a. | <input checked="" type="checkbox"/> | A check in the amount of \$ <u>1170.00</u> to cover the above fees is enclosed. |
| b. | <input type="checkbox"/> | Please charge my Deposit Account No. 25-0120 in the amount of \$ to cover the above fees. A duplicate copy of this sheet is enclosed. |
| c. | <input checked="" type="checkbox"/> | The Commissioner is hereby authorized to charge any additional fees which may be required by 37 CFR 1.16 and 1.17, or credit any overpayment to Deposit Account No. 25-0120 . A duplicate copy of this sheet is enclosed. |

SEND ALL CORRESPONDENCE TO.

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December 31, 2001

Customer Number: 000466

By Thomas W. Perkins
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100.1955.06702
10/019565
531 Rec'd PCT. 31 DEC 2001

PATENTS

IN THE UNITED STATES PATENT AND TRADEMARK OFFICE

In re application of
Nils Olof HÅKANSSON et al.
Serial No. (unknown)
Filed herewith
OIL SEPARATOR FOR SMALL PARTICLES

PRELIMINARY AMENDMENT

Commissioner for Patents

Washington, D.C. 20231

Sir:

Prior to the first Official Action and calculation of the filing fee, please substitute Claims 1-9 as originally filed, which appear on pages 6-7, with Claims 1-8 as filed in the Article 34 amendment of June 18, 2001. The pages containing Claims 1-8 are marked "AMENDED SHEET" and are attached hereto. Following the insertion of Claims 1-8, please amend these claims as follows:

IN THE CLAIMS:

Please amend claims 4-5 and 7-8 as follows:

--4. (Amended) Device according to claim 2, **characterized in** that the end walls have two axially spaced walls (19, 37 and 20, 38, respectively) which, together with intermediate radially directed wall portions (39), define said passages (35, 36).

5. (Amended) Device according to claim 1, **characterized in** that the drive means are formed by a liquid turbine (30).

Nils Olof HÅKANSSON et al.

7. (Amended) Device according to claim 1, **characterized in** that the extent of the channel layers radially amounts to circa 20% of the radius of the first container (17).

8. (Amended) Device according to claim 1, **characterized in** that the radius of the first container (17) is circa 100 mm and its length circa 200 mm, that the container is designed to be driven at a rotational speed of circa 3 000 rpm, that the radial dimension (r) of the channels (34) is less than 1 mm, and that the outermost channel layer is located at a maximum radial distance from the rotational axis.--

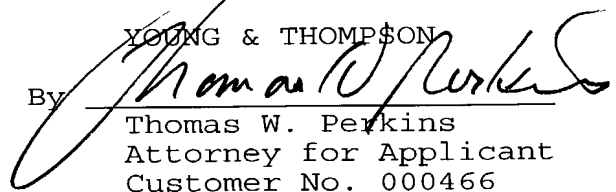
R E M A R K S

The above changes in the claims merely place this national phase application in the same condition as it was during Chapter II of the international phase, with the multiple dependencies being removed. Following entry of this amendment by substitution of the pages, only claims 1-8 remain pending in this application. Claims 4-5 and 7-8 have been amended to correct multiple dependencies. Attached hereto is a marked-up version of the changes made to the claims by the current amendment. The attached page is captioned **"VERSION WITH MARKINGS TO SHOW CHANGES MADE"**.

Respectfully submitted,

YOUNG & THOMPSON

By



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December 31, 2001

"VERSION WITH MARKINGS TO SHOW CHANGES MADE"

4. ~~(Amended)~~ Device according to claim 2 ~~or 3~~, **characterized in** that the end walls have two axially spaced walls (19, 37 and 20, 38, respectively) which, together with intermediate radially directed wall portions (39), define said passages (35, 36).

5. ~~(Amended)~~ Device according to ~~one of claims 1-4~~, **claim 1, characterized in** that the drive means are formed by a liquid turbine (30).

7. ~~(Amended)~~ Device according to ~~one of claims 1-6~~, **claim 1, characterized in** that the extent of the channel layers radially amounts to circa 20% of the radius of the first container (17).

8. ~~(Amended)~~ Device according to ~~one of claims 1-7~~, **claim 1, characterized in** that the radius of the first container (17) is circa 100 mm and its length circa 200 mm, that the container is designed to be driven at a rotational speed of circa 3 000 rpm, that the radial dimension (r) of the channels (34) is less than 1 mm, and that the outermost channel layer is located at a maximum radial distance from the rotational axis.

Claims

18 -06- 2001

1. Device for separating small drops of liquid from a flowing gas, containing liquid mist, comprising a first container (17), formed of a cylindrical lateral wall (18) and two opposite end walls (19, 20), said first container having an inlet (23) in one end wall for the gas/mist, a first outlet (24) in its other end wall for the gas and a second outlet (25) at or in the vicinity of the second end wall for liquid separated out of the gas, a second container (28) with means (26, 27) for rotatably mounting the first container in the second container rotationally symmetrically in relation to said inlet and said first outlet, and drive means (30) for rotation of the first container in the second container, wherein the inlet (23) and the outlets (24, 25) communicate with each other via a plurality of layers of peripherally spaced, narrow channels (34) extending in the direction of the rotational axis and being disposed at a radial distance from the rotational axis, said narrow channels having a radial dimension (r) being adapted to their length, their radial distance to the rotational axis, the flow velocity of the gas and the rotational speed, so that at least the major portion of the liquid drops in the mist will have time to be deposited on the channel walls before they reach the outlet, **characterized in that** the inlet (23) of the first container (17) is joined to a ventilation conduit from a crankcase of an internal combustion engine (1), and its outlet (24) is joined to an inlet air conduit to the engine.

2. Device according to claim 1, **characterized in that** the inlet (23) has an inlet opening placed concentrically with the rotational axis of the first container, said inlet opening leading to a plurality of radial inlet passages (35) leading out to inlets of the channels (34), and that the channels have outlets opening, firstly, into a plurality of radial outlet passages (36), through which the gas can flow into an outlet opening (24) placed concentrically with said rotational axis, and, secondly, into a plurality of outlet openings (25), through which liquid deposited under the effect of the centrifugal force on the walls of the channels, can flow radially outwards to a space between the first container and the second container.

3. Device according to claim 2, **characterized in** that the containers (17, 28) are oriented with the rotational axis vertical so that liquid separated out can flow, by the force of gravity, to the inlet end and out through a liquid outlet (42) from the second container (28).

5

4. Device according to claim 2 or 3, **characterized in** that the end walls have two axially spaced walls (19, 37 and 20, 38, respectively) which, together with intermediate radially directed wall portions (39), define said passages (35, 36).

10

5. Device according to one of claims 1-4, **characterized in** that the drive means are formed by a liquid turbine (30).

15

6. Device according to claim 5, **characterized in** that the end walls (19, 20) are joined to rotatably mounted hollow shafts (21, 22), the interiors of which forming inlet and outlet channels, respectively (23 and 24, respectively), and in that a turbine wheel (30) is fixed to one of said shafts.

20

7. Device according to one of claims 1-6, **characterized in** that the extent of the channel layers radially amounts to circa 20% of the radius of the first container (17).

25

8. Device according to one of claims 1-7, **characterized in** that the radius of the first container (17) is circa 100 mm and its length circa 200 mm, that the container is designed to be driven at a rotational speed of circa 3 000 rpm, that the radial dimension (r) of the channels (34) is less than 1 mm, and that the outermost channel layer is located at a maximum radial distance from the rotational axis.

Oil separator for small particles

The present invention relates to a device for separating small drops of liquid from a flowing gas containing liquid mist, comprising a first container, formed of a cylindrical lateral wall and two opposite end walls, said first container having an inlet in one end wall for the gas/mist, a first outlet in its other end wall for the gas and a second outlet at or in the vicinity of the second end wall for liquid separated out of the gas, a second container with means for rotatably mounting the first container in the second container rotationally symmetrically in relation to said inlet and said first outlet, and drive means for rotation of the first container in the second container.

It is a known fact that it is not possible to achieve piston ring seals between the pistons and the surrounding cylinder walls of the cylinders in an internal combustion engine, which seal off the combustion chambers 100% against the engine crankcase.

A certain small amount of combustion gases, called here blow-by gas, thus always flows past the piston rings and down into the engine crankcase. In order to prevent excessive pressure caused by the blow-by gas in the crankcase, the crankcase must be ventilated and the gas drawn off leaving only a low overpressure in the crankcase.

Two types of crankcase ventilation are used, viz. either open or closed ventilation. An engine with open ventilation can, quite simply, have a downwardly directed tube connected to the valve cover, for example, and which opens into the surrounding atmosphere. In an engine with closed crankcase ventilation, the blow-by gases from the crankcase are led to the engine intake conduit and are mixed with the intake air.

When evacuating blow-by gases, it has, up to now, been unavoidable that a certain amount of oil mist will accompany them. The amount of oil being carried with the gas depends on the placement of the ventilation outlet and any filters or oil traps in the crankcase ventilation. Regardless of whether the engine has open or closed crankcase ventilation, it is desirable to keep the amount of oil in the evacuated

blow-by gas at a minimum. In the first case, it is to minimize the effect on the environment and to keep the engine oil consumption at a low level. In the second case, it is to prevent oil deposits on or in the components of the engine intake system, e.g. oil deposits on the impeller blades of the compressor of a turbo-charged engine or oil deposits in the charge air cooler in engines with charge air cooling.

A number of different devices to separate oil from blow-by gases from internal combustion engines are known. Among them are various types of baffle or screen systems, metal wire or textile fibre filters as well as cyclones and centrifuges. With the aid of baffle and screen systems, it has up to now been possible to separate oil drops down to a diameter of circa 10 μm , with metal wire filters and cyclones down to circa 1.5 μm and with multi-layer textile fibre filters, e.g. fleece filters, down to circa 0.4 μm . Baffle and screen systems have, in principle, an unlimited lifetime and do not produce any pressure losses, but do not provide sufficiently effective oil separation, since circa 80% of the oil mass in the oil mist in the blow-by gases consists of oil drops with a diameter which is less than 1.5 μm . Previously known metal wire filters and cyclones thus are not sufficient either for achieving effective oil separation despite the fact that they have acceptable lifetimes and provide limited pressure losses. Multi-layer fleece filters remain, but these have limited lifetimes and therefore must be replaced often, leading to high costs. They also produce significant pressure losses.

The purpose of the present invention is to achieve a device of the type described by way of introduction, i.e. a separator of centrifugal type, which is particularly, but not exclusively, intended for separation of oil mist drops from blow-by gases from internal combustion engines and which can separate liquid drops of less than 1 μm without having the disadvantage of a multi-layer fleece filter, for example.

This is achieved according to the invention by virtue of the fact that the inlet and the outlets communicate with each other via a plurality of layers of peripherally spaced,

narrow channels extending in the direction of the rotational axis and being disposed at a radial distance from the rotational axis, said narrow channels having a radial dimension being adapted to their length, their radial distance to the rotational axis, the flow velocity of the gas and the rotational speed, so that at least the major portion of the liquid drops in the mist will have time to be deposited on the channel walls before they reach the outlet.

Tests performed have demonstrated that in a preferred embodiment particularly designed for separating oil mist from blow-by gases, with a suitable selection of the above-mentioned parameters, more than 90% of all the oil mist in the gases was separated out. The gas velocity, the distance which the oil drops must move radially outwards to strike the channel wall, and the radial acceleration determine in this case the required length of stay for the drops in the channels. This length of stay determines in turn the required channel length. It has been shown that the diameter (of circular channels) should not exceed, but should preferably be less than 1 mm for the separator to have reasonable dimensions and a reasonable rotational speed.

The invention will be described in more detail below with reference to examples shown in the accompanying drawings, where Fig. 1 shows a schematic representation of an engine installation with a device according to the invention, Fig. 2 is a schematic longitudinal section through one embodiment of a device according to the invention, and Fig. 3 is an enlarged cross-section along the line III-III in Fig. 2.

1 in Fig. 1 designates the engine block of a six-cylinder, four-stroke diesel engine, and 2 designates a gearbox with clutch, coupled to the engine crankshaft. The engine is supercharged by a turbo compressor 3, which has a turbine 4 coupled to the engine exhaust manifold 5, and a compressor 6 coupled to the inlet manifold 7 via a charge air cooler 8. The suction side of the compressor 6 is coupled via an inlet conduit 9 to an air filter 10.

In the crankcase of the engine 1, a screen separator 11 and a baffle separator 12, known per se and only schematically indicated here, are arranged. A conduit 13 with a pressure regulator valve 14 and a pressure sensor 15 connect the engine crankcase with the inlet conduit 9 via an oil mist separator 16 according to the invention, which is shown in more detail in Fig. 2.

The separator 16 comprises an inner container 17, which is formed of a cylinder 18 and two end walls 19 and 20, each having a central opening in which a tubular shaft 21 and 22, respectively, is fixed. The interior channel 23 of the tubular shaft 21 forms an inlet for combustion gases and oil mist, while the channel 24 of the tubular shaft 22 forms an outlet for exhaust. Peripherally spaced openings 25 in the cylinder 18 form outlets for separated-out oil. The tubular shafts 21, 22 are mounted in bearings 26, 27 of a stationary outer container 28, which has a lower portion 29 forming a turbine housing. The lower tubular shaft 21 extends through the turbine housing 29 and carries a turbine rotor 30. The housing 29 has an inlet 31 and an outlet 32 for oil from the engine lubrication system. When the engine is in operation and lubricant is pumped through the housing 29, the turbine rotor 30 drives the shaft 21 so that the inner container 17 rotates in the outer container 28.

A cylindrical package 33 consisting of a plurality of layers of axial channels 34 is fixed to the lateral wall 18 of the inner container 17. It is also possible to integrate the cylindrical lateral wall 18 in the channel package 34. The channels 34 are open at their ends and communicate with radial passages 35, 36, which are defined between the respective end wall 19, 20, a pair of conical wall elements 37, 38 and intermediate, radially directed, peripherally evenly spaced intermediate walls 39. An enlarged portion of a channel package 33 with ten layers is shown in cross section in Fig. 3. The channels 34 are greatly enlarged. In realistic tests with an inner container with a channel length of circa 200 mm, an inner container radius of circa 100 mm and a rotational speed of circa 3 000 rpm, the radial dimension "r" of the channels should be slightly less than 1 mm, in order for there to be enough time for the

smallest oil drops to strike the channel walls before the gases have flown out through the channels and out into the outlet passages 36.

5 In the example shown in Fig. 3, the channels 34 in the channel package 33 are formed of alternating smooth and folded sheet metal cylinders 40 and 41, respectively, i.e. corresponding to the construction of corrugated cardboard. The channel package can also have a solid body, e.g. a ceramic body with drilled or cast cylindrical channels. The radial extent of the channel package amounts to approximately 20% of the radius of the first container. If the percentage is greater, there is a risk
10 that the radially innermost channels will be so close to the rotational axis that the centrifugal force on the smallest drops will be too low to accelerate them out to the channel wall.

15 The oil, which is transported by the combustion gases through the channels 34 and under the influence of the centrifugal force as the inner container rotates, strikes the channel walls, is ejected through the openings 25, as indicated by the solid arrows, and runs down into the space between the containers to an outlet 42. From there the oil is led back to the engine crankcase. The combustion gases free of oil mist flow out through the upper tubular shaft 22, as indicated by the dashed arrow.

Claims

1 8 -06- 2001

1. Device for separating small drops of liquid from a flowing gas, containing liquid mist, comprising a first container (17), formed of a cylindrical lateral wall (18) and two opposite end walls (19, 20), said first container having an inlet (23) in one end wall for the gas/mist, a first outlet (24) in its other end wall for the gas and a second outlet (25) at or in the vicinity of the second end wall for liquid separated out of the gas, a second container (28) with means (26, 27) for rotatably mounting the first container in the second container rotationally symmetrically in relation to said inlet and said first outlet, and drive means (30) for rotation of the first container in the second container, wherein the inlet (23) and the outlets (24, 25) communicate with each other via a plurality of layers of peripherally spaced, narrow channels (34) extending in the direction of the rotational axis and being disposed at a radial distance from the rotational axis, said narrow channels having a radial dimension (r) being adapted to their length, their radial distance to the rotational axis, the flow velocity of the gas and the rotational speed, so that at least the major portion of the liquid drops in the mist will have time to be deposited on the channel walls before they reach the outlet, **characterized in** that the inlet (23) of the first container (17) is joined to a ventilation conduit from a crankcase of an internal combustion engine (1), and its outlet (24) is joined to an inlet air conduit to the engine.

2. Device according to claim 1, **characterized in** that the inlet (23) has an inlet opening placed concentrically with the rotational axis of the first container, said inlet opening leading to a plurality of radial inlet passages (35) leading out to inlets of the channels (34), and that the channels have outlets opening, firstly, into a plurality of radial outlet passages (36), through which the gas can flow into an outlet opening (24) placed concentrically with said rotational axis, and, secondly, into a plurality of outlet openings (25), through which liquid deposited under the effect of the centrifugal force on the walls of the channels, can flow radially outwards to a space between the first container and the second container.

3. Device according to claim 2, **characterized in** that the containers (17, 28) are oriented with the rotational axis vertical so that liquid separated out can flow, by the force of gravity, to the inlet end and out through a liquid outlet (42) from the second container (28).

5

4. Device according to claim 2 or 3, **characterized in** that the end walls have two axially spaced walls (19, 37 and 20, 38, respectively) which, together with intermediate radially directed wall portions (39), define said passages (35, 36).

10 5. Device according to one of claims 1-4, **characterized in** that the drive means are formed by a liquid turbine (30).

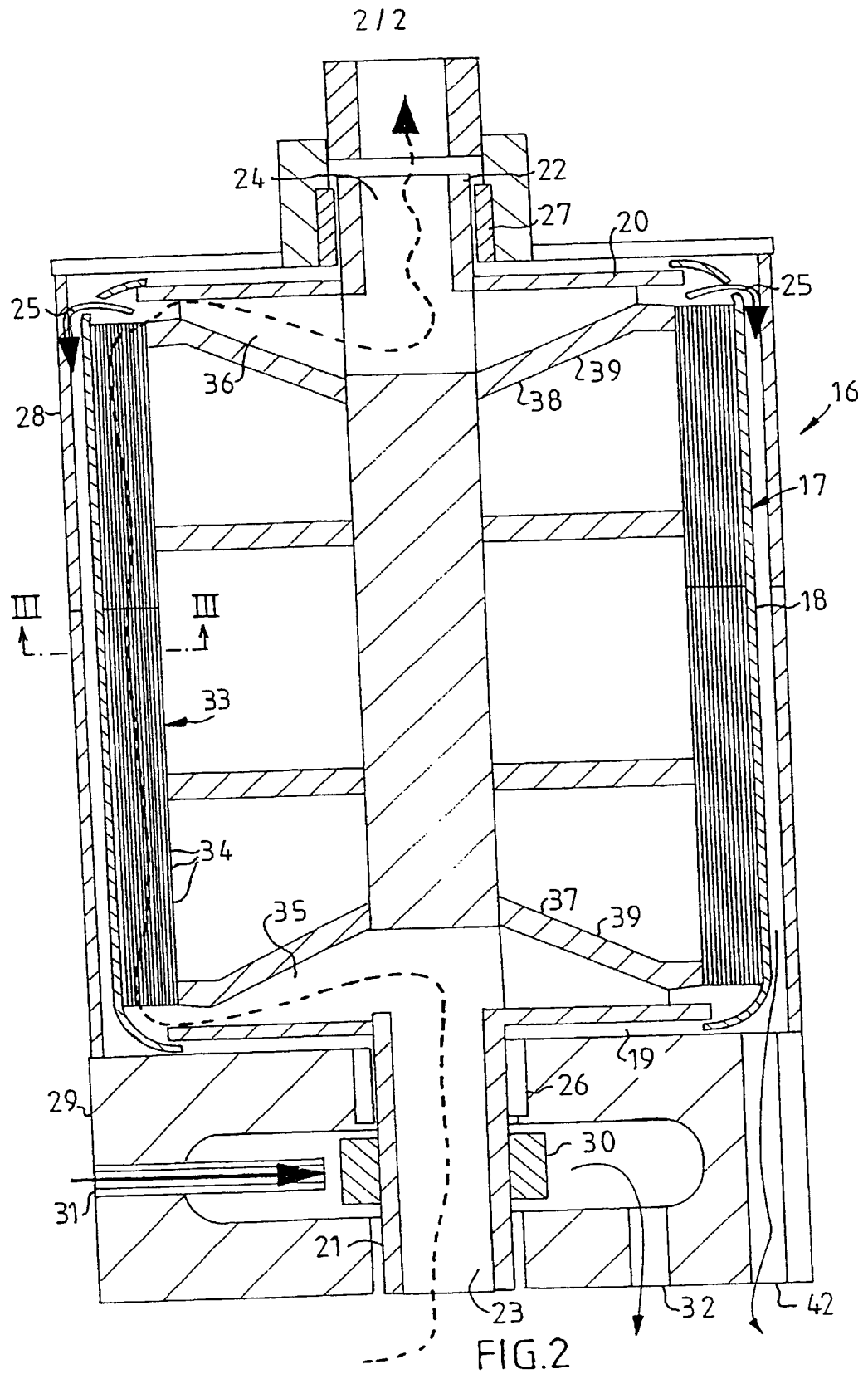
15 6. Device according to claim 5, **characterized in** that the end walls (19, 20) are joined to rotatably mounted hollow shafts (21, 22), the interiors of which forming inlet and outlet channels, respectively (23 and 24, respectively), and in that a turbine wheel (30) is fixed to one of said shafts.

20 7. Device according to one of claims 1-6, **characterized in** that the extent of the channel layers radially amounts to circa 20% of the radius of the first container (17).

25 8. Device according to one of claims 1-7, **characterized in** that the radius of the first container (17) is circa 100 mm and its length circa 200 mm, that the container is designed to be driven at a rotational speed of circa 3 000 rpm, that the radial dimension (r) of the channels (34) is less than 1 mm, and that the outermost channel layer is located at a maximum radial distance from the rotational axis.

Abstract

Device for separating oil mist from blow-by gases from an internal combustion engine, comprising a rotating cylindrical container (17) with a central inlet (23) for
5 gases with oil mist at one end and a central outlet (24) for gases and peripheral
outlets (25) for separated oil at the other end. Between the inlet and the outlet,
narrow channels (34) extend in several layers, spaced from the axis of rotation,
through which the gas with oil drops flows. Under the influence of the centrifugal
force the drops of oil are moved at the same time radially, the rotational speed and
10 the length and radial dimension of the channels being adapted to each other so that
the oil drops strike the channel walls before flowing out of the channels.



SUBSTITUTE SHEET (RULE 26)

COMBINED DECLARATION AND POWER OF ATTORNEY

As a below named inventor, I hereby declare that

My residence, post office address and citizenship are as stated below next to my name.

I believe I am the original, first and sole inventor (if only one name is listed below) or an original, first and joint inventor (if plural names are listed below) of the subject matter which is claimed and for which a patent is sought on the invention entitled: OIL SEPARATOR FOR SMALL PARTICLES

the specification of which: (check one)

REGULAR OR DESIGN APPLICATION

- ☐ is attached hereto.
- ☒ was filed on December 31, 2001 as application Serial No. _____ and was amended on _____ (if applicable).

PCT FILED APPLICATION ENTERING NATIONAL STAGE

- ☒ was described and claimed in International application No. PCT/SE00/01394 filed on June 30, 2000 and as amended on _____ (if any).

I hereby state that I have reviewed and understand the contents of the above-identified specification, including the claims, as amended by any amendment referred to above.

I acknowledge the duty to disclose information which is material to patentability as defined in Title 37, Code of Federal Regulations, §1.56.

PRIORITY CLAIM

I hereby claim foreign priority benefits under 35 USC 119 of any foreign application(s) for patent or inventor's certificate listed below and have also identified below any foreign application for patent or inventor's certificate having a filing date before that of the application on which priority is claimed.

PRIOR FOREIGN APPLICATION(S)

Country	Application Number	Date of Filing (day, month, year)	Priority Claimed
SWEDEN	9902490-3	30 June 1999	Yes

I hereby claim the benefit under Title 35, United States Code §119(e) of any United States provisional patent application(s) listed below:

Application No.	Filing Date	Status (patented, pending abandoned)
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(Complete this part only if this is a continuing application.)

I hereby claim the benefit under 35 USC 120 of any United States application(s) listed below and, insofar as the subject matter of each of the claims of this application is not disclosed in the prior United States application in the manner provided by the first paragraph of 35 USC 112, I acknowledge the duty to disclose information which is material to patentability as defined in Title 37 Code of Federal Regulations §1.56 which became available between the filing date of the prior application and the national or PCT international filing date of this application:

Application No.	Filing Date	Status (patented, pending abandoned)
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POWER OF ATTORNEY

The undersigned hereby authorizes the U.S. attorney or agent named herein to accept and follow instructions from **ALBIHNS STOCKHOLM AB** as to any action to be taken in the Patent and Trademark Office regarding this application without direct communication between the U.S. attorney or agent and the undersigned. In the event of a change in the persons from whom instructions may be taken, the U.S. attorney or agent named herein will be so notified by the undersigned.

As a named inventor, I hereby appoint the registered patent attorneys represented by Customer No. **000466** to prosecute this application and transact all business in the Patent and Trademark Office connected therewith, including: **Robert J. PATCH, Reg. No. 17,355, Andrew J. PATCH, Reg. No. 32,925, Robert F. HARGEST, Reg. No. 25,590, Benoît CASTEL, Reg. No. 35,041, Thomas W. PERKINS, Reg. No. 33,027, Roland E. LONG, Jr., Reg. No. 41,949, and Eric JENSEN, Reg. No. 37,855,**

c/o YOUNG & THOMPSON,
Second Floor,
745 South 23rd Street,
Arlington, Virginia 22202.



00466

PATENT TRADEMARK OFFICE

Address all telephone calls to Young & Thompson at 703/521-2297. Telefax: 703/685-0573.

I hereby declare that all statements made herein of my own knowledge are true and that all statements made on information and belief are believed to be true; and further that these statements were made with the knowledge that willful false statements and the like so made are punishable by fine or imprisonment, or both under Section 1001 of Title 18 of the United States Code and that such willful false statements may jeopardize the validity of the application or any patent issued thereon.

Full name of sole or first inventor: 100 Nils Olof HÅKANSSON

Inventor's signature: [Signature]

Date: 12/2 2002

Residence: Stenkullen, SWEDEN

SEX

Citizenship: SWEDEN

Post Office Address: Flintakroken 12, Stenkullen, SWEDEN

Full name of second joint inventor, if any: 200 Per PERSSON

Inventor's signature: Per Persson

Date: 29/1 2002

Residence: Partille, SWEDEN

SEX

Citizenship: SWEDEN

Post Office Address: Fornfyndsvägen 18, S-433 41, Partille, SWEDEN

Full name of third joint inventor, if any: _____

Inventor's signature: _____

Date: _____

Residence: _____

Citizenship: _____

Post Office Address: _____

Full name of fourth joint inventor, if any: _____

Inventor's signature: _____

Date: _____

Residence: _____

Citizenship: _____

Post Office Address: _____